



**Fraunhofer** Institut  
Solare Energiesysteme

Short Report: KTB Nr. 2003-31

## Efficiency curve according to EN 12975-2

**Brand Name:**

T2

**Responsible for Testing:**

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**Date:**

11th November 2003

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Test facility certified by DIN CERTCO





## 1 Test Center

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### 3 Collector efficiency curve

#### 3.1 Test method

Outdoor, steady state according to EN 12975-2:  
Thermal solar systems and components, solar collectors, test methods

#### 3.2 Description of the calculation

The functional dependence of the collector efficiency on the meteorological and system operation values can be represented by the following mathematical equation:

$$\eta_{(G,(t_m-t_a))} = \eta_0 - a_{1a} \frac{t_m - t_a}{G} - a_{2a} \frac{(t_m - t_a)^2}{G} \quad (1)$$

(based on aperture area)

with:  $t_m = \frac{(t_e + t_{in})}{2}$

where:  $G$  = global irradiance on the collector area ( $W/m^2$ )  
 $t_{in}$  = collector inlet temperature ( $^{\circ}C$ )  
 $t_e$  = collector outlet temperature ( $^{\circ}C$ )  
 $t_a$  = ambient temperature ( $^{\circ}C$ )

The coefficients  $\eta_0$ ,  $a_{1a}$  and  $a_{2a}$  have the following meaning:

$\eta_0$ : Efficiency without heat losses, which means that the mean collector fluid temperature is equal to the ambient temperature:

$$\frac{(t_{in} + t_e)}{2} = t_a$$

The coefficients  $a_{1a}$  and  $a_{2a}$  describe the heat loss of the collector. The temperature dependency of the collector heat loss is described by:

$$a_{1a} + a_{2a} * (t_m - t_a)$$

### 3.3 Instantaneous efficiency curve based on aperture and absorber area and mean temperature of heat transfer fluid

Boundary conditions:

Test method:	outdoor, steady state
Latitude:	48.0°
Longitude:	7.8°
Collector tilt:	tracked between 40° and 50°
Collector azimuth:	tracked

Test conditions:

Mean irradiation :	854 W/m <sup>2</sup>
Mean wind speed:	3 m/s
Mean flow rate:	145 kg/h
Kind of fluid:	water
Period:	November 2003

$G$ [W/m <sup>2</sup> ]	$G_d/G$ [-]	$m$ [kg/h]	$t_{in}$ [°C]	$t_e$ [°C]	$t_e - t_{in}$ [K]	$t_m$ [°C]	$t_a$ [°C]	$t_m - t_a$ [K]	$(t_m - t_a)/G$ [K m <sup>2</sup> /W]	$\eta_a$ [-]
922	0.08	146.5	7.39	15.76	8.37	11.57	11.98	-0.41	-0.0004	0.770
917	0.08	146.6	7.37	15.70	8.33	11.54	11.52	0.01	0.0000	0.771
914	0.08	146.6	7.36	15.64	8.28	11.50	11.65	-0.15	-0.0002	0.769
919	0.08	146.6	7.37	15.70	8.34	11.53	11.97	-0.43	-0.0005	0.770
793	0.08	143.8	34.87	41.08	6.20	37.98	11.61	26.37	0.0332	0.649
755	0.08	143.7	34.90	40.78	5.89	37.84	11.57	26.27	0.0348	0.646
732	0.08	143.7	34.90	40.54	5.65	37.72	11.42	26.30	0.0359	0.640
721	0.08	143.8	34.89	40.45	5.56	37.67	11.34	26.34	0.0365	0.639
844	0.08	144.7	62.99	67.95	4.96	65.47	9.43	56.04	0.0664	0.491
855	0.08	144.6	62.98	68.08	5.10	65.53	9.70	55.84	0.0653	0.499
871	0.08	144.6	63.00	68.29	5.28	65.64	10.10	55.55	0.0638	0.507
882	0.08	144.9	63.01	68.39	5.38	65.70	10.22	55.48	0.0629	0.511
890	0.10	144.7	91.31	95.06	3.75	93.19	8.53	84.66	0.0952	0.354
883	0.10	144.8	91.29	95.02	3.73	93.16	8.22	84.94	0.0962	0.355
879	0.10	144.6	91.31	95.05	3.73	93.18	8.79	84.39	0.0960	0.356
879	0.09	144.6	91.33	95.10	3.77	93.22	8.94	84.28	0.0959	0.360

Table 1: Data of determined efficiency points

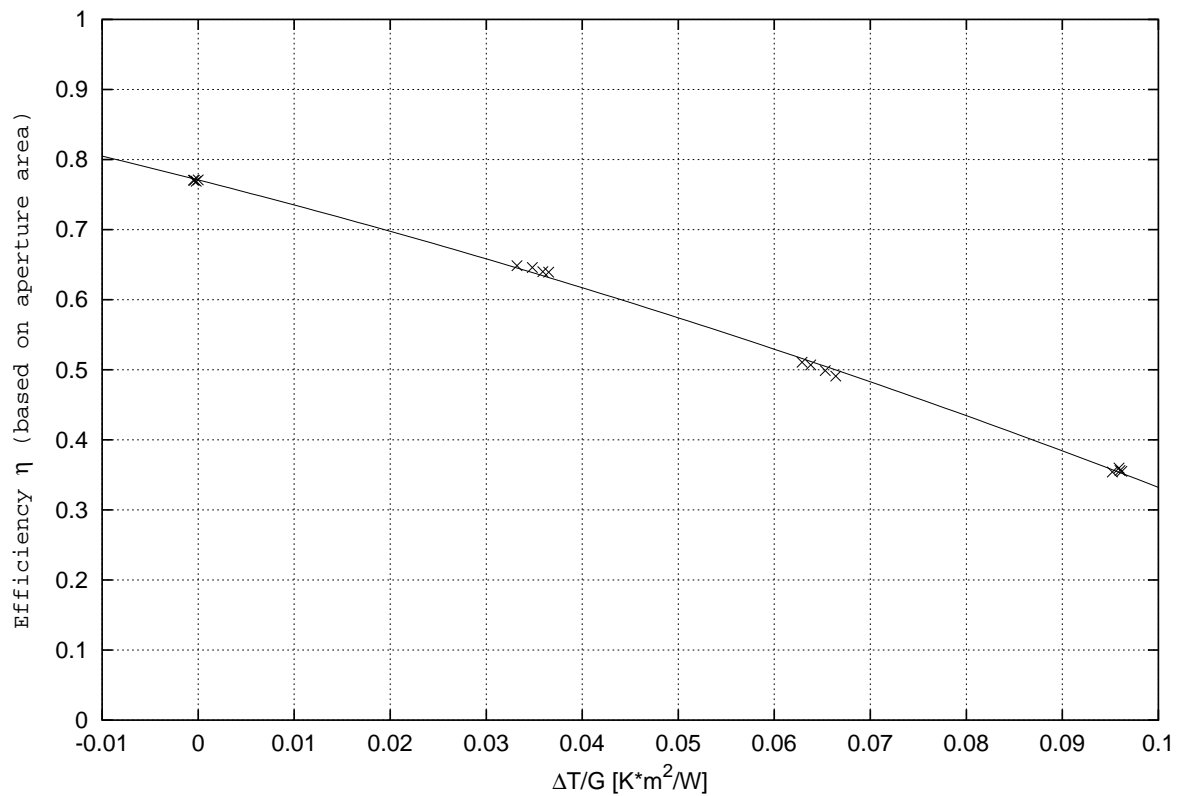


Figure 1: Efficiency curve with measurement points based on aperture area 2.013 m<sup>2</sup>

Results based on aperture area 2.013 m<sup>2</sup>:

$$\begin{aligned} \eta_{0a} &= 0.771 \\ a_{1a} &= 3.529 \text{ W/m}^2\text{K} \\ a_{2a} &= 0.0097 \text{ W/m}^2\text{K}^2 \end{aligned}$$



### 3.4 Calculation of the efficiency parameters to other reference areas

The efficiency parameters can be calculated to other reference areas according the following procedure (Example absorber area  $A_A$ ):

$$\eta_{0A} = A_a/A_A * \eta_{0a}$$

$$a_{1A} = A_a/A_A * a_{1a}$$

$$a_{2A} = A_a/A_A * a_{2a}$$

This report is not a complete report according EN-12975-2.

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Freiburg, 11th November 2003

Fraunhofer-Institute for Solar Energy Systems ISE

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